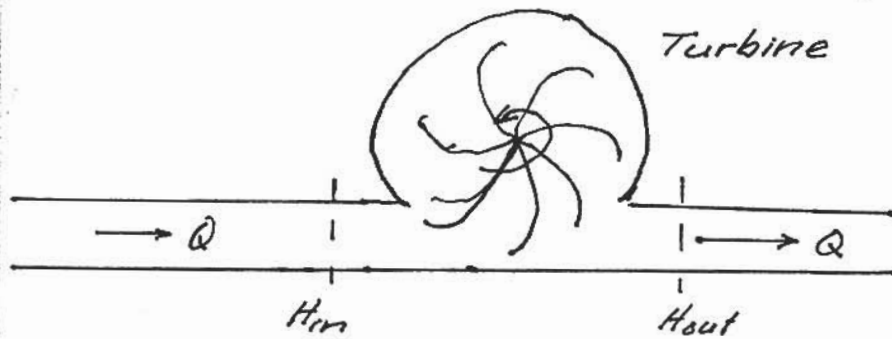


LECTURE # 19

1.060 ENGINEERING MECHANICS II

TURBINES

A turbine may be regarded as an "inverse pump" in that extracts energy (e.g. producing electricity) from the mechanical energy of a flow



H_{in} and H_{out} are obtained from pipe flow analyses identical to those performed for pumps.

Energy considerations then give:

$$\dot{E}_{in} - \dot{E}_{out} = \rho g Q (H_{in} - H_{out}) = \rho g Q H_T$$

where

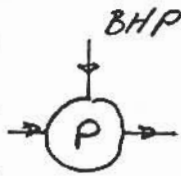
$$H_T = H_{in} - H_{out} = \text{Turbine Head}$$

Conversion of the flow power, $\rho g Q H_T$, to an alternative form of power is associated with a loss, so for turbines

$$\underline{\text{BHP}} = \text{Power Produced by Turbine} = \eta \rho g Q H_T \quad \eta \leq 1$$

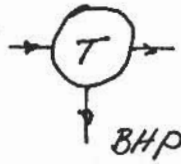
$\eta = \text{efficiency}$

So what happens to the Energy Lost?



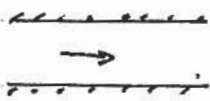
$$\eta \text{ BHP} = \rho g Q H_p$$

$(1-\eta)\text{BHP}$ "lost"



$$\eta \rho g Q H_T = \text{BHP}$$

$(1-\eta)\rho g Q H_T$ "lost"

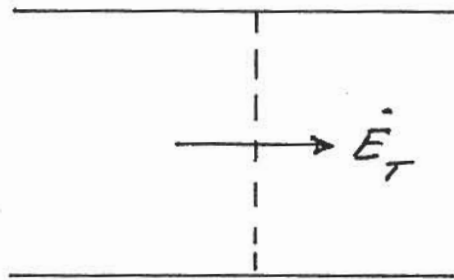


$$\rho g Q H_1 = \rho g Q H_2 + \rho g Q \Delta H$$

$\rho g Q \Delta H$ "lost"

What actually happens to the "lost" energy?

ENERGY REVISITED



$$\dot{E}_{\text{Total}} = \dot{E}_{\text{mech}} + \dot{E}_{\text{internal}} = \dot{E}_{\text{org.}} + \dot{E}_{\text{disorg}}$$

$$\dot{E}_{\text{mech}} = \dot{E}_{\text{org.}} = \rho g Q H$$

$$\dot{E}_{\text{internal}} = \dot{E}_{\text{disorg}} = \rho Q \hat{u}$$

$\rho \hat{u}$ = internal (disorganized - thermal-heat) energy per unit volume of fluid. =

$$\rho \hat{u} = \rho C_p T = \rho C_v T \quad \text{[for incompressible fluid, water]}$$

T = absolute Temperature ($^{\circ}\text{K} = 273.15 + ^{\circ}\text{C}$)

C_p = specific heat @ constant pressure } same for
 C_v = " " " " volume } an incompressible fluid.

$$[\rho C_p T] = \frac{\text{kg}}{\text{m}^3} [C_p]^{\circ}\text{K} = \frac{\text{Energy}}{\text{Volume}} = \frac{\text{Nm}}{\text{m}^3} = \frac{(\text{kg} \frac{\text{m}}{\text{s}^2}) \text{m}}{\text{m}^3}$$

$$[C_p] = \frac{\text{m}^2}{\text{s}^2 \cdot ^{\circ}\text{K}}$$

$$\text{For water: } C_p = C_v = 4,210 \frac{\text{m}^2}{\text{s}^2 \cdot ^{\circ}\text{K}}$$

ENERGY (TOTAL) CONSERVATION

$$\dot{E}_{\text{Total}} = \dot{E}_T = \rho Q [gH + \hat{u}]$$

$$\dot{E}_{T,\text{in}} - \dot{E}_{T,\text{out}} \neq \underbrace{\dot{H}_{\text{add}} - \dot{H}_{\text{loss}}}_{= 0} = 0$$

Net inflow from boundaries $\vec{q} \cdot \vec{n} = 0$

If insulated pipe, then

$$\rho g Q (H_1 - H_2) = \rho Q (\hat{u}_2 - \hat{u}_1)$$

Rate of dissipation = Rate of production
of mechanical energy of Thermal (internal) energy

$$g(H_1 - H_2) = g \Delta H = \hat{u}_2 - \hat{u}_1 = C_p(T_2 - T_1)$$

or

$$\Delta T = T_2 - T_1 = \frac{g \Delta H}{C_p}$$

Loss of head causes an increase in temperature!
 Material constants, e.g. ρ and ν , are functions of temperature; but we have treated them as constants.
 Is this justified?

Pipe Flow Example

$V = 1 \text{ m/s}$; $D = 0.025 \text{ m (1")}$; $f = 0.02$; $L = 1,000 \text{ m}$.

$$\Delta H = \Delta H_f = f \frac{L}{D} \frac{V^2}{2g}$$

$$\Delta T = T_2 - T_1 = \frac{g \Delta H}{C_p} = \frac{f(L/D)V^2}{2C_p} = \frac{0.02 \cdot 1000 \cdot 1}{2 \cdot 4200} = 0.1 \text{ K}$$

Negligible change in temperature. Indeed neglect of temperature variations is justified!