
Prob. 19.21 - Cantilevered beam

The stress σ_x has no dependency on the material properties, and is not influenced by material viscoelasticity.

The correspondence-principle recipe starts by putting the deflection relation in the Laplace plane (C is the compliance operator):

```
> v_bar := (x^2 * (3*L-x) / (6*Ix)) * C * F_bar;
```

$$v_{\text{bar}} := \frac{1}{6} \frac{x^2 (3L-x) C F_{\text{bar}}}{Ix}$$

Transform of load:

```
> with(inttrans): F_bar := laplace(F * Heaviside(t), t, s);
```

$$F_{\text{bar}} := \frac{F}{s}$$

Compliance operator:

```
> C := Cg + Cv / (tau * (s + 1/tau));
```

$$C := C_g + \frac{C_v}{\tau \left(s + \frac{1}{\tau} \right)}$$

Invert to get deflection in time plane:

```
> v(t) := invlaplace(v_bar, s, t);
```

$$v(t) := \frac{1}{6} \frac{x^2 (3L-x) F \left(-C_v e^{\left(-\frac{t}{\tau} \right)} + C_g + C_v \right)}{Ix}$$

Simplifying manually to standard form:

$$v(x,t) = \frac{x^2 (3L-x)}{6I} \cdot F \cdot \left[C_g + C_v (1 - e^{-t/\tau}) \right]$$

Superposition approach: write load and compliance as time functions:

```
> F := (t) -> F[0] * Heaviside(t);
```

$$F := t \rightarrow F_0 \text{Heaviside}(t)$$

```
> C[crp] := (t) -> C[g] + C[v] * (1 - exp(-t/tau));
```

$$C_{\text{crp}} := t \rightarrow C_g + C_v \left(1 - e^{\left(-\frac{t}{\tau} \right)} \right)$$

Superposition integral:

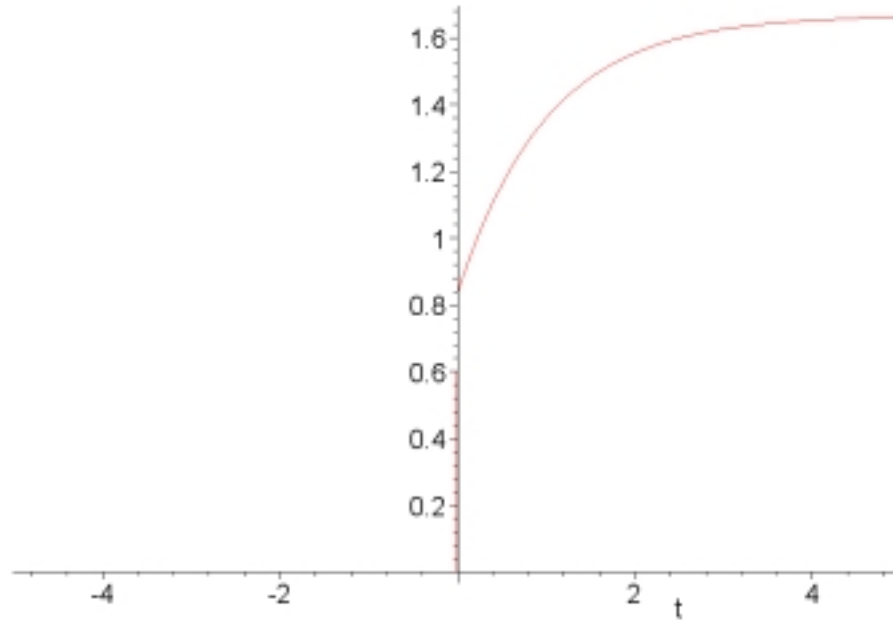
```
> v(t) := (x^2 * (3*L-x) / (6*Ix)) * int(C[crp](t-xi) * diff(F(xi), xi), xi = -infinity..t);
```

$$v(t) := -\frac{1}{6} \frac{x^2 (3L - x) F_0 \text{Heaviside}(t) \left(-C_g - C_v + C_v e^{\left(-\frac{t}{\tau}\right)} \right)}{Ix}$$

This can be reduced to the same form obtained previously.

Examine deflection function for arbitrary choice of parameters:

```
> plot(subs({L=2,x=1,F[0]=1,C[g]=1,C[v]=1,tau=1,Ix=1},v(t)),t=-5..5)
;
```



Prob. 19.22 - Rigid die

Define Poisson (N) and tensile modulus (EE) operators in terms of dilatation (K) and shear (G) operators:

```
> N:=(3*K-2*G)/(6*K+2*G);EE:=(9*G*K)/(3*K+G);
```

$$N := \frac{3K - 2G}{6K + 2G}$$

$$EE := 9 \frac{GK}{3K + G}$$

SLS expressions for G and K:

```
> G:=Gr+((Gg-Gr)*s)/(s+(1/tau_G));
```

$$G := Gr + \frac{(Gg - Gr)s}{s + \frac{1}{\tau_G}}$$

```
> K:=Kr+((Kg-Kr)*s)/(s+(1/tau_K));
```

$$K := Kr + \frac{(Kg - Kr) s}{s + \frac{1}{\tau_K}}$$

Pick model parameters from Fig. 17. Relaxation times (τ_K and τ_G) are those times at which the relaxation has dropped (1/e) of its total value.

```
> Digits:=20:Gg:=8.8*10^8:Gr:=2.4*10^5:tau_G:=.001:
> Kg:=6.2*10^9:Kr:=1.7*10^9:tau_K:=.0005:
> sigybar:=sigma[y]/s;
```

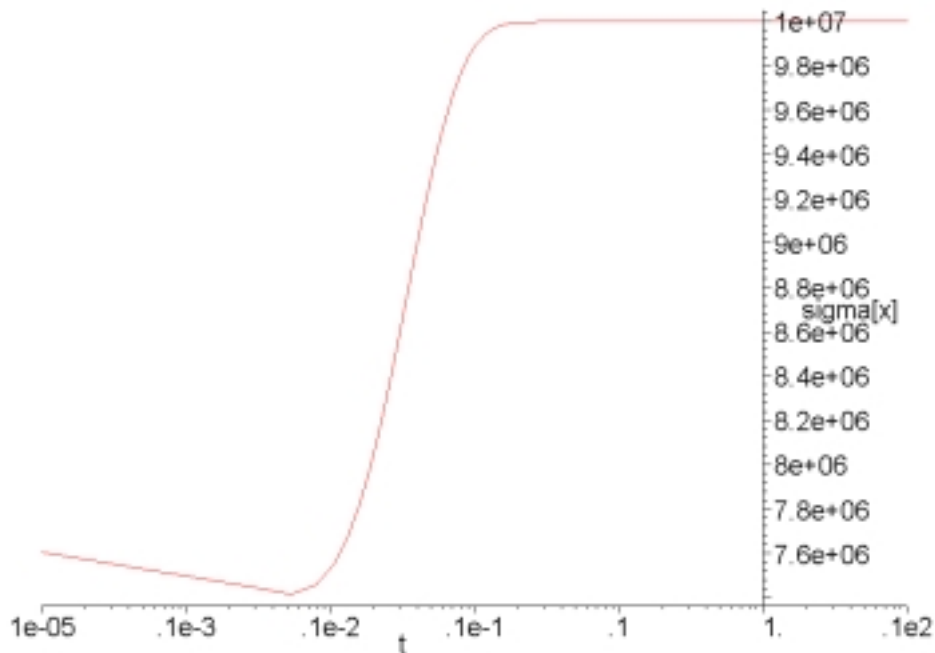
$$\text{sigybar} := \frac{\sigma_y}{s}$$

Transverse stress:

```
> sigxbar:=N*sigybar/(1-N):
> sigma[y]:=10*10^6;
```

$$\sigma_y := 10000000$$

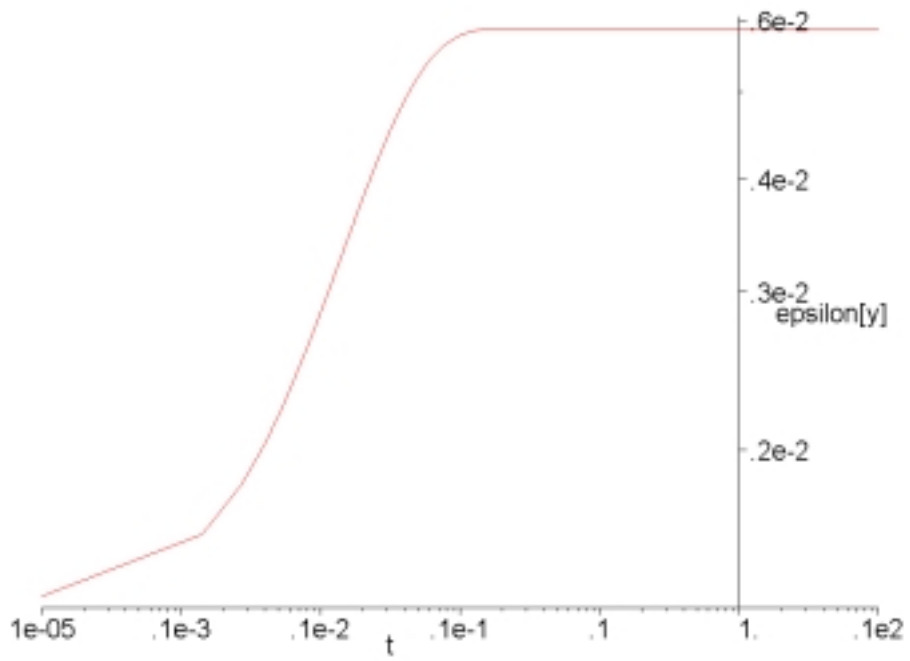
```
> sigma[x]:=invlaplace(sigxbar,s,t):
> with(plots):semilogplot(sigma[x],t=10^(-5)..10,labels=[t,`sigma[x]`],numpoints=5000);
```



Note that the transverse stress becomes equal to the vertical stress (i.e. the stress state becomes hydrostatic) as the relaxation completes and the material approaches a rubbery state.

Vertical strain:

```
> epsybar:=(1+N)*(1-2*N)*sigybar/(EE*(1-N)):
> epsilon[y]:=invlaplace(epsybar,s,t):
> loglogplot(epsilon[y],t=10^(-5)..10,labels=[t,`epsilon[y]`],numpoints=5000);
```



We might expect the $(1-2\nu)$ factor to drive the strain to zero as ν approaches 0.5, but the tensile modulus is also dropping substantially, and the strain is observed to rise. The material does not become fully rubbery, and maintains a finite compressibility.