

2.76 / 2.760 Lecture 13: Interfaces/forces

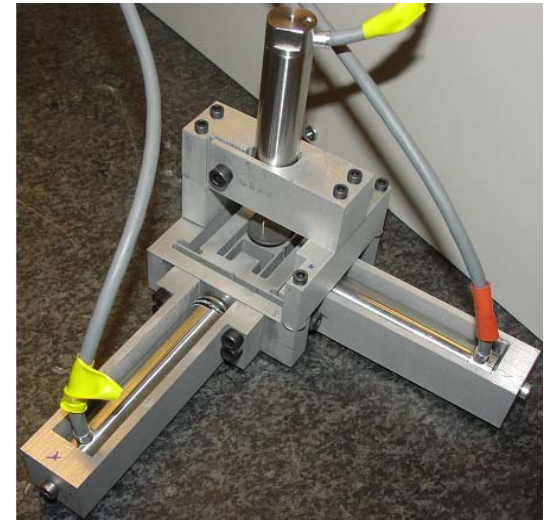
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Purpose of today

Bolted joints (pointers)

Kinematic coupling experiment

- Repeatability
- Stiffness
- Metrology issues
- Perspective for use in STM

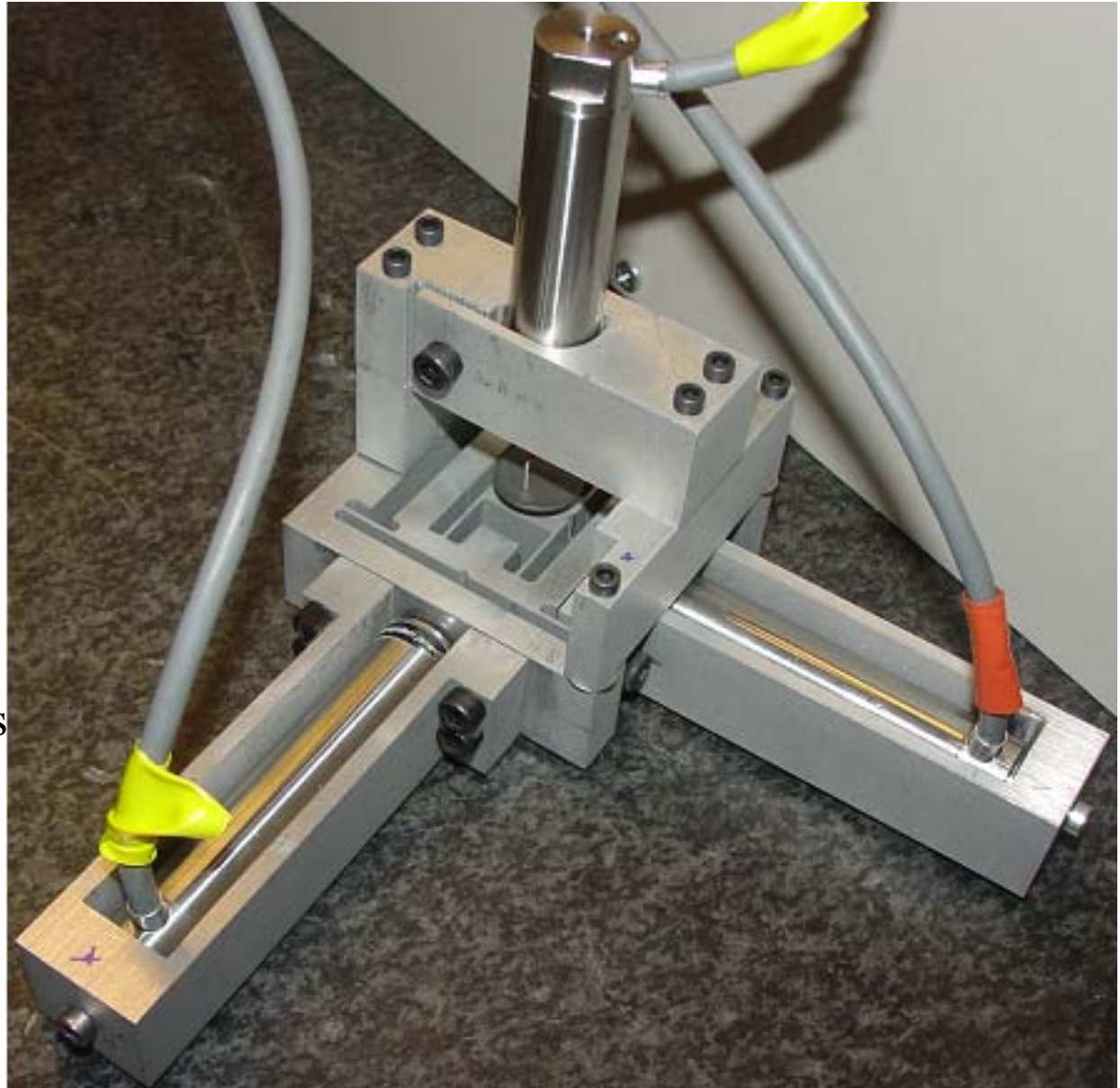


Project questions/design help

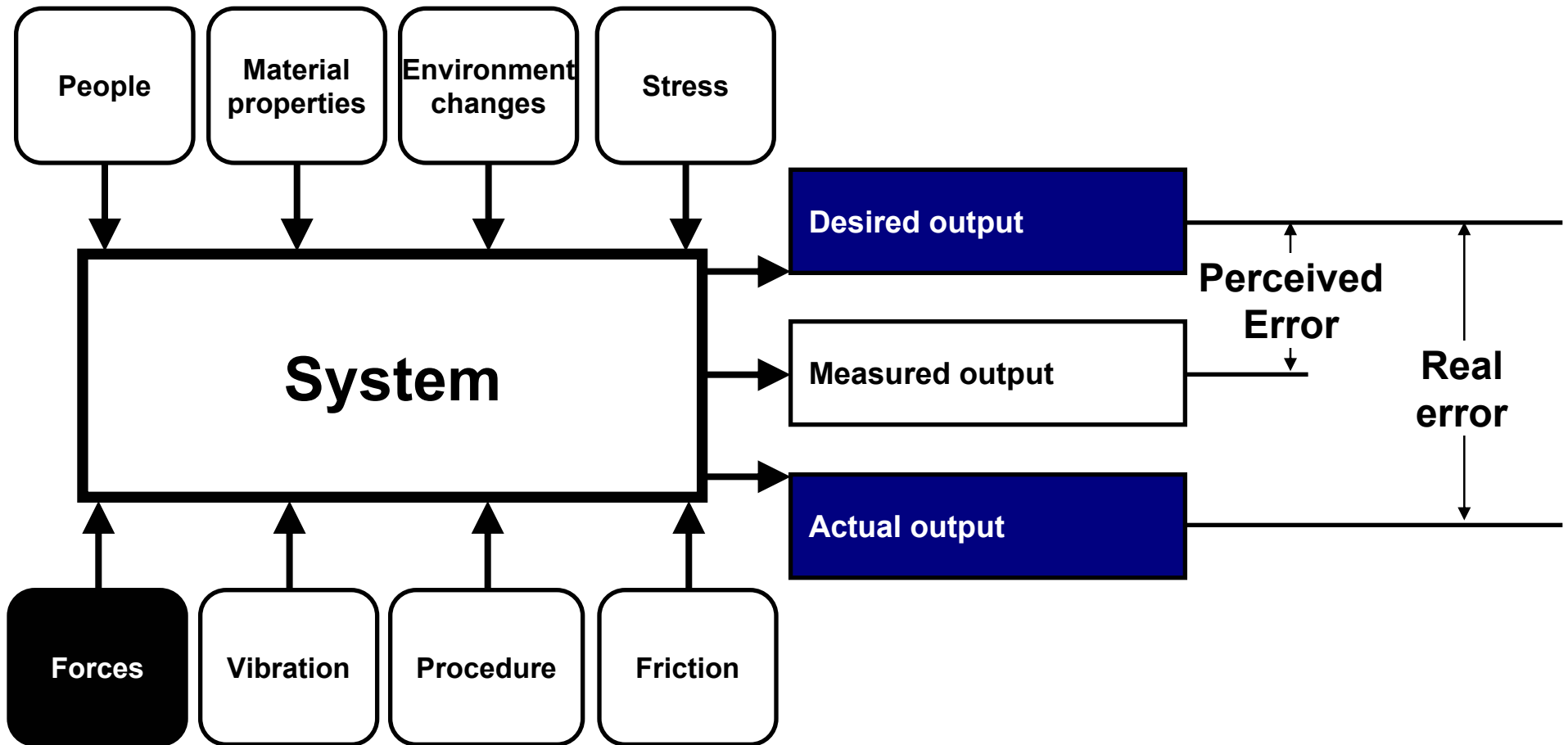
Bolted stiffness and stability

Qualitatively determine:

- ❑ What is good?
- ❑ What is bad?
- ❑ Think about:
 - Static loads
 - “Dynamic” loads
 - Bending moments
 - Mass

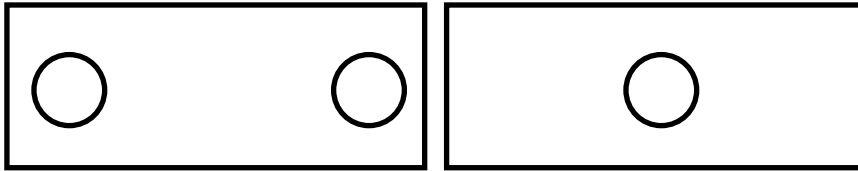


Minimize variation

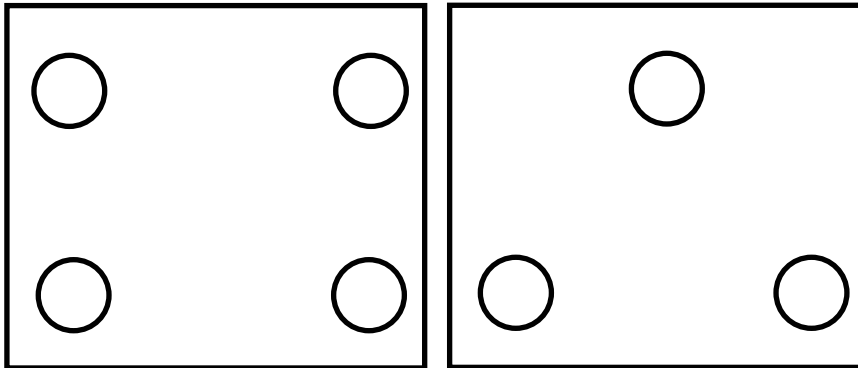


Bolted joint topology

A

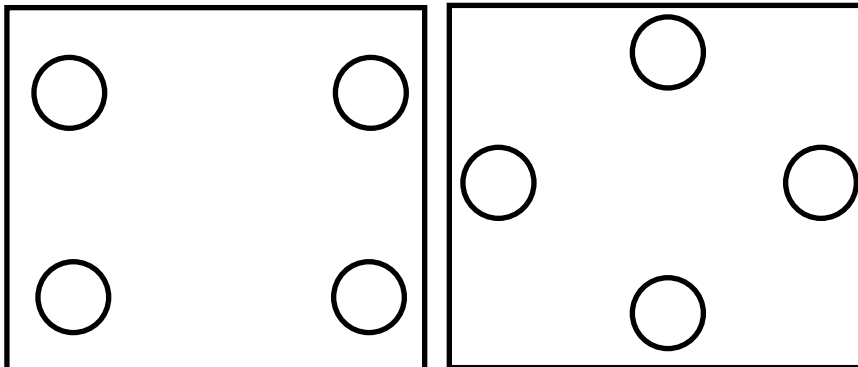


Singular

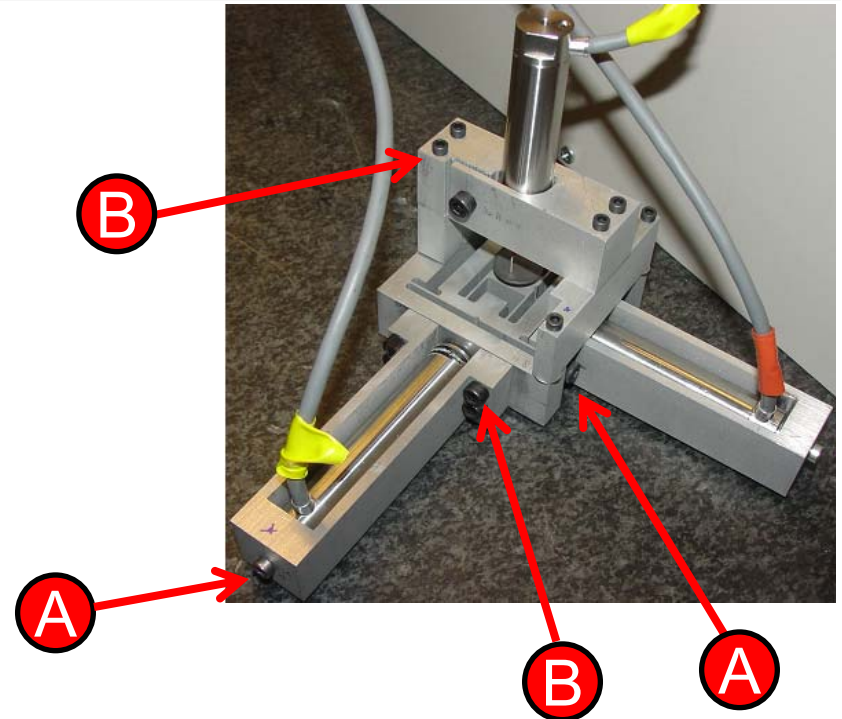


Over constraint

B



In-plane Moment stiffness



Bolt physics

Rough estimate:

$$E_{in} = E_{out} + E_{stored} + E_{"generated"}$$

$$F_{bolt} = A_{cross} \cdot \frac{\sigma_{yield}}{FS}$$

$$T_{applied} \cdot \theta = T_{f-threads} \cdot \theta + T_{f-head} \cdot \theta + E_{stretch} + 0$$

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$$T_{applied} \cdot \theta = (C_1 \cdot r_{bolt} \cdot \mu_{f-threads} \cdot F_{bolt}) \cdot \theta + (C_2 \cdot r_{bolt} \cdot \mu_{f-head} \cdot F_{bolt}) \cdot \theta + \int_0^{\delta} F_{bolt} \cdot dx$$

Interface variation

On average get about 30 – 50% of contact at mechanical interfaces

These “little springs” reduce joint stiffness

Stiffness important for dynamic disturbances

Use potting to stiffen up the joint interface...

**Calculate joint stiffness: Shigley/Mischke
Machine Design**

KC Experiment

Study: Sensitivity due to mechanical interface

- Step 1: Setup – pay attention up front
- Step 2: Test repeatability
 - Non-lubricated (35 mates)
 - Lubricated (35 mates)
- Step 3: Test stiffness
 - Preload levels 1, 2 and 3 (note moment arms....)
- Step 4: Compare theory (spread sheet) and measured
 - $E = 30 \text{ Mpsi}$ $\nu = 0.28$ $R_i = \text{measured}$

Questions to think about

- How do (not should) lateral & axial stiffness scale with preload?
- Plate stiffness (sweet spot) to prevent metrology errors?
- How repeatable is your bio-unit preload?
- Who has the best repeatability?
- What are the implications for your STM?