

# **Chapter 11    Design of Low Friction Surfaces**

# Design of Low Friction Surfaces (no lubricant allowed)

- Consider the task of creating low friction surfaces for sliding applications.

**FR<sub>1</sub> = Support the normal load**

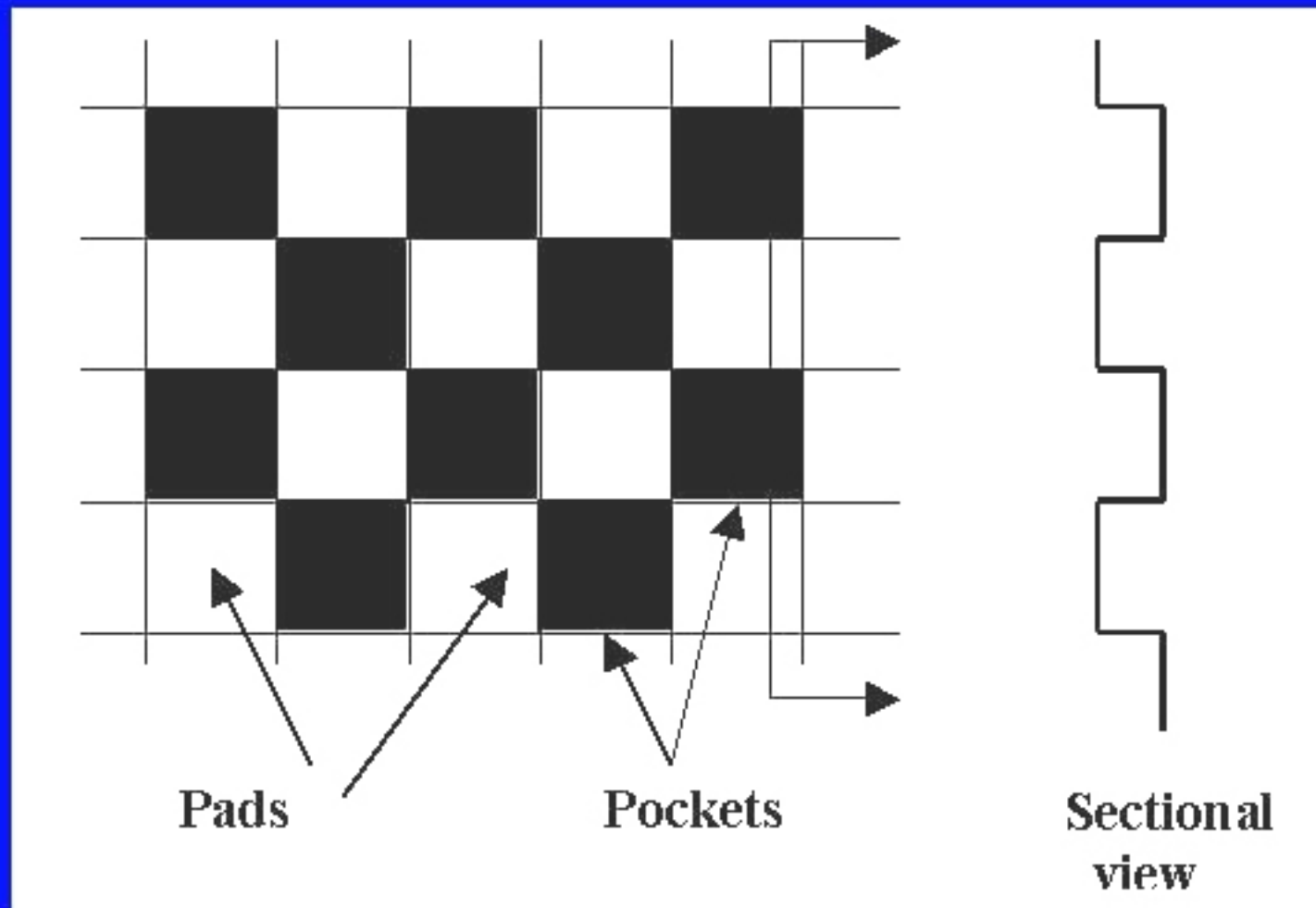
**FR<sub>2</sub> = Prevent particle generation**

**FR<sub>3</sub> = Prevent particle agglomeration**

**FR<sub>4</sub> = Remove wear particles from the interface**

- The constraint is that lubricants cannot be used.

# Design of Low Friction Surfaces



# Design of Low Friction Surfaces

- The design parameters (DPs) for the undulated surface that can satisfy the FRs are as follows:

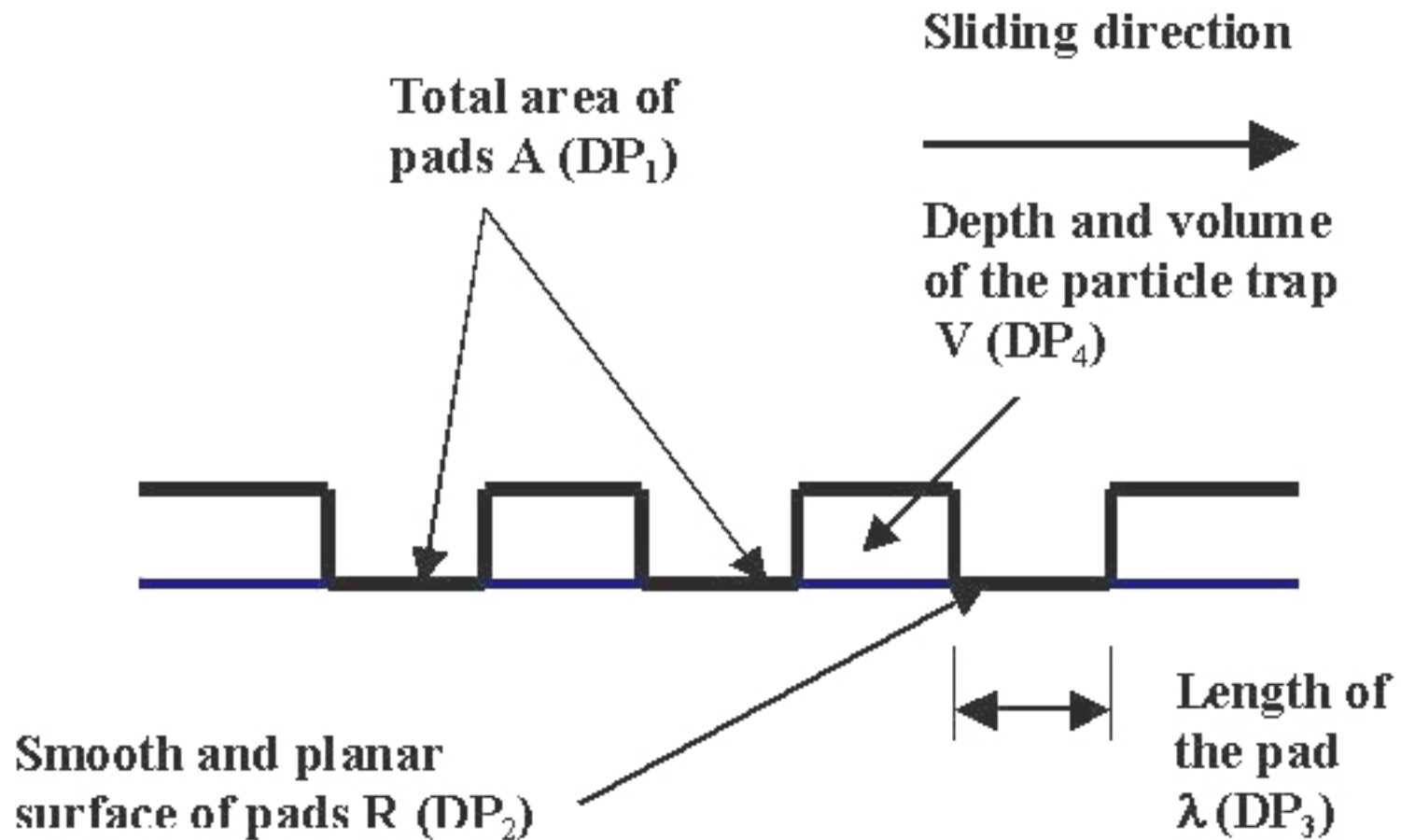
**DP<sub>1</sub> = Total contact area of the pad, A**

**DP<sub>2</sub> = Roughness of the planar surface of pads, R**

**DP<sub>3</sub> = Length of the pad in the sliding direction,  $\lambda$**

**DP<sub>4</sub> = Volume and depth of the pocket for wear particles, V**

# Design of Low Friction Surfaces



# Design Equation for Undulated Surface

$$\begin{Bmatrix} FR_1 \\ FR_2 \\ FR_3 \\ FR_4 \end{Bmatrix} = \begin{bmatrix} X & 0 & 0 & 0 \\ 0 & X & x & 0 \\ 0 & 0 & X & 0 \\ 0 & 0 & 0 & X \end{bmatrix} \begin{Bmatrix} DP_1 \\ DP_2 \\ DP_3 \\ DP_4 \end{Bmatrix} = \begin{Bmatrix} X & 0 & 0 & 0 \\ 0 & X & x & 0 \\ 0 & 0 & X & 0 \\ 0 & 0 & 0 & X \end{Bmatrix} \begin{Bmatrix} A \\ R \\ \lambda \\ V \end{Bmatrix}$$

# Lubricated Low Friction Surface

FR<sub>1</sub> = Support the normal load

FR<sub>2</sub> = Prevent particle generation

FR<sub>3</sub> = Prevent particle agglomeration

FR<sub>4</sub> = Remove wear particles from the interface

# Design of Lubricated Low Friction Surfaces

- FRs are

$FR_1$  = Support the normal load

$FR_2$  = Prevent particle generation

$FR_3$  = Prevent particle agglomeration

$FR_4$  = Remove wear particles from the interface



# Design of Lubricated Low Friction Surfaces

- **DPs are**

**DP<sub>1</sub> = Total contact area of the pad, A**

**DP<sub>2</sub> = Roughness of the planar surface of pads,  
R**

**DP<sub>3</sub> = Boundary lubricant, Lub**

**DP<sub>4</sub> = Volume and depth of the pocket for wear  
particles, V**

# Geometrically Constrained System

<b>Material</b>	<b>Mean Roughness Ra (<math>\mu\text{m}</math>)</b>	<b>Rockwell Hardness</b>
M50 Ball	0.050	C-Scale 62
304 Stainless Steel Blocks	0.263	B-Scale 80
MoS2 coated Blocks	0.361	-
AFSL 200 coated Blocks	0.388	-

# Geometrically Constrained System

## Surface properties of shafts and bushings

<b>Material</b>	<b>Mean Roughness Ra(<math>\mu\text{m}</math>)</b>	<b>Hardness (Rockwell)</b>
Hardened Steel Bushing	0.255	C-scale 68
304 Stainless steel Shaft	0.263	B-Scale 80
MoS2 coated shafts	0.361	-

# Geometrically Constrained System

Steady state friction coefficients of uncoated and coated surfaces obtained from pin-on-disk tests. Normal Load = 50g

Specimen	Smooth	Undulated
Uncoated	0.7	0.18
MoS <sub>2</sub> Coated	0.24	0.18
AFSL 200 Coated	0.125	0.11

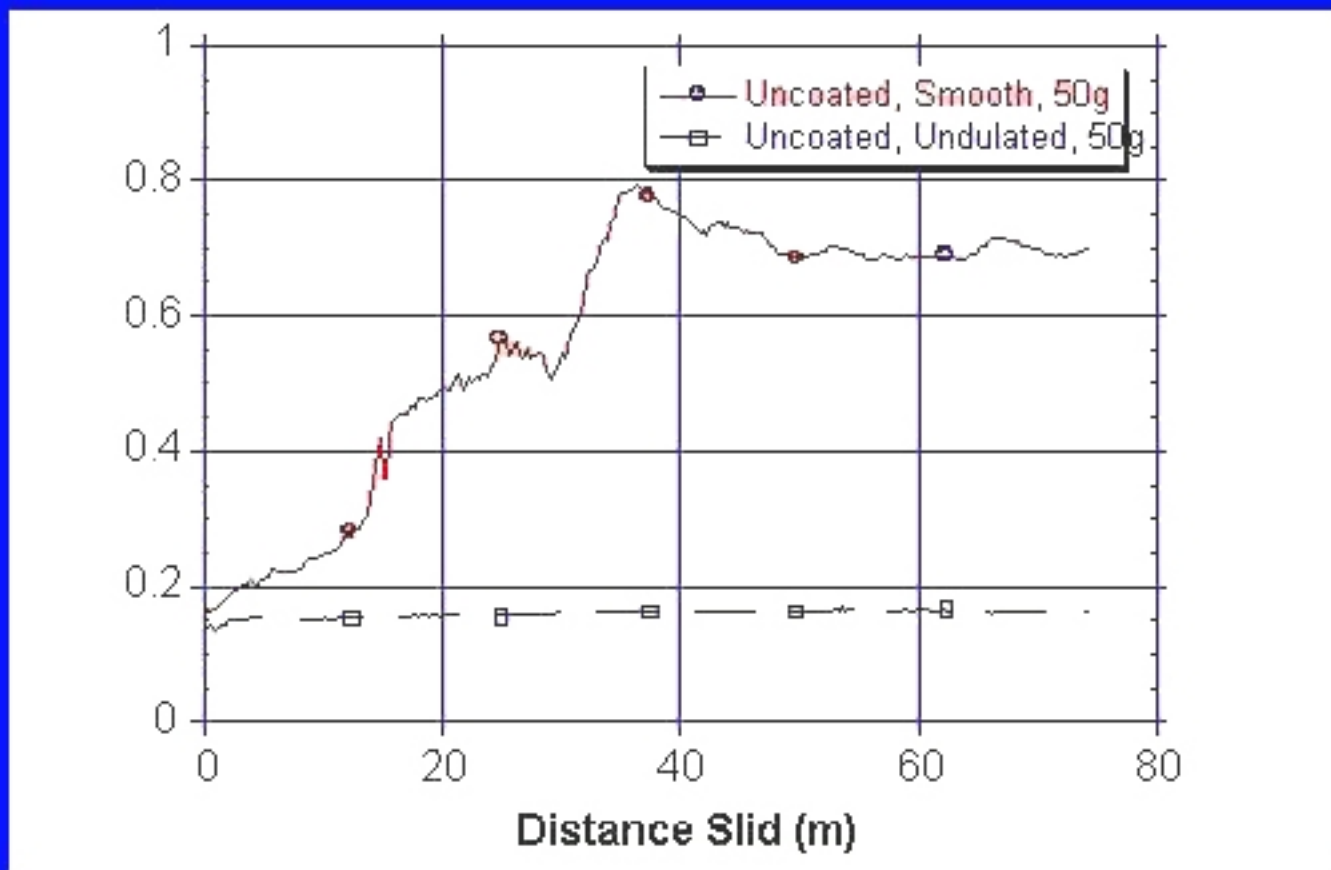
# Geometrically Constrained System

Steady state friction coefficients of uncoated and coated surfaces obtained from pin-on-disk tests. Normal Load = 200g

Specimen	Smooth	Undulated
Uncoated	0.75	0.42
MoS <sub>2</sub> Coated	0.175	0.125
AFSL 200 Coated	0.100	0.075

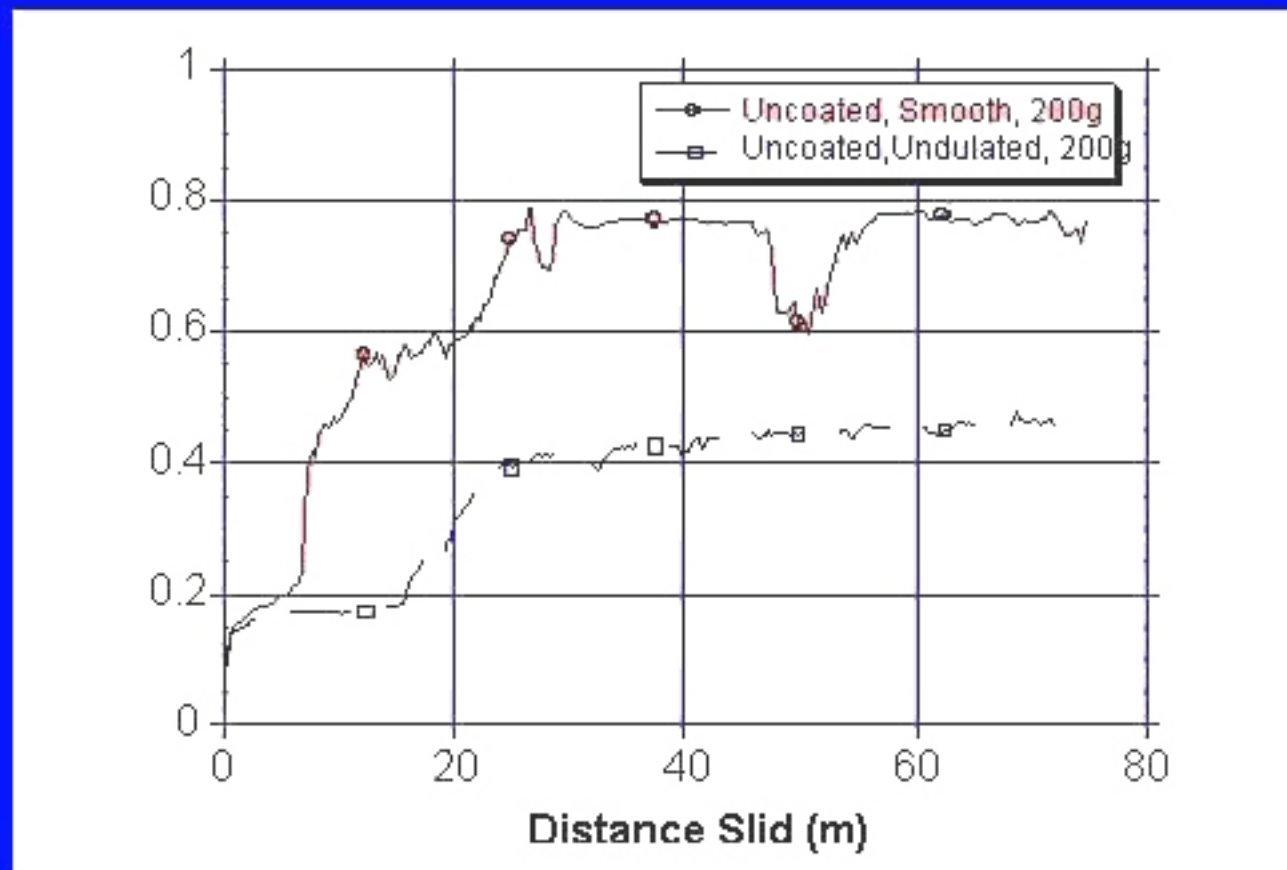
# Geometrically Constrained System

Pin-on-disk experiment with uncoated stainless steel 304 sliding against M50 steel ball under normal load of 50 g.



# Geometrically Constrained System

Pin-on-disk experiment with uncoated stainless steel 304 sliding against M50 steel ball under normal load of 200 g.



# Microscale Undulated Surfaces

Microstructured surface of silicon. The textured dimension of the undulation — the width of the line -- of (a) and (b) is 5  $\mu\text{m}$ . The corresponding dimension of (c) and (d) is 50  $\mu\text{m}$ . [Cha and Kim, 2000]

Figures removed for copyright reasons.

Source: Kim, D. E., K. H. Cha, and I. H. Sung. "Design of Surface Micro-structures for Friction Control in Micro-systems Applications." *Annals of the CIRP* 51, no. 1 (2002): 495-498.



# Microscale Undulated Surfaces

The worn surface of Si<sub>3</sub>N<sub>4</sub> coated flat surface and the undulated surface. The undulated surface did not wear much. Wear particles were in the pocket. [Cha and Kim, 2000]

Figures removed for copyright reasons.

Source: Kim, D. E., K. H. Cha, and I. H. Sung. "Design of Surface Micro-structures for Friction Control in Micro-systems Applications." *Annals of the CIRP* 51, no. 1 (2002): 495-498.

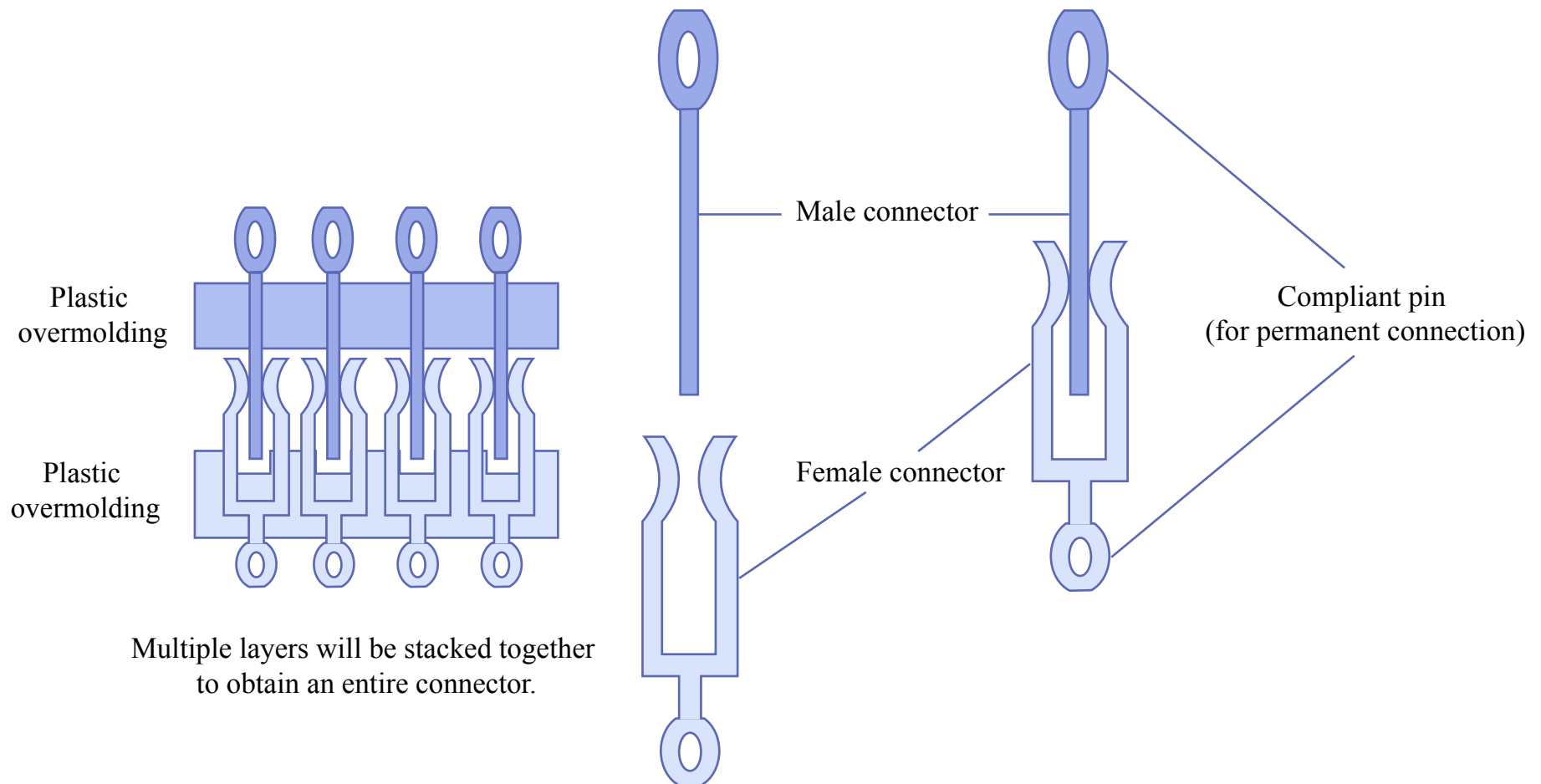
# Microscale Undulated Surfaces

The friction coefficients of aluminum coated flat surface and microscale textured undulated surfaces at two different loads. The one on the left is for 1 gf and the one on right is for 5 gf. [Cha and Kim, 2000]

Figures removed for copyright reasons.

Source: Kim, D. E., K. H. Cha, and I. H. Sung. "Design of Surface Micro-structures for Friction Control in Micro-systems Applications." *Annals of the CIRP* 51, no. 1 (2002): 495-498.

# Conventional Electrical Connectors



## **FRs of an Electrical Connector**

**FR1 = Mechanically connect and disconnect electrical terminals**

**FR2 = Control contact resistance (should be less than 20mΩ)**

**FR3 = Prevent the cross-talk (i.e., interference) between the connections**

**Subject to the following constraints (Cs):**

**C1 = Low cost**

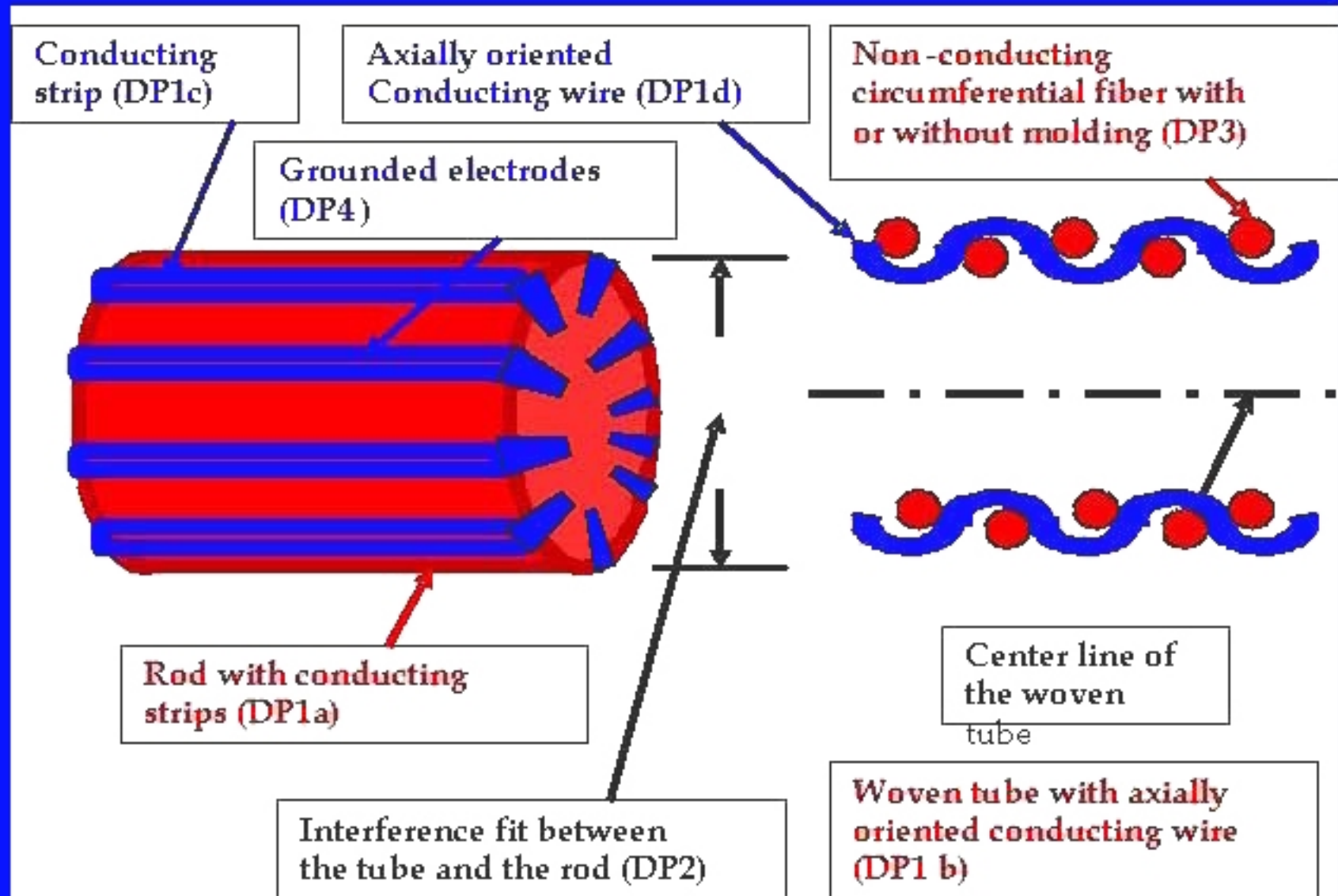
**C2 = Ease of use**

**C3 = Long life (> million cycles)**

**C4 = Maximum temperature rise of 30 °C**

**C5 = Low insertion force**

# Physical Embodiment of an Electrical Connector



## **DPs of an Electrical Connector**

**DP1 = Cylindrical assembly of the woven tube and the pin**

**DP2 = Locally compliant electric contact**

**DP3 = Number of conducting wires**

**Decomposition of FR1 (Mechanically connect and disconnect electrical terminals) and DP1 (Cylindrical assembly of the woven tube and the pin)**

**FR11 = Align the rod axially inside the tube**

**FR12 = Locate the axial position of the rod in the tube**

**FR13 = Guide the pin**

**DP11 = Long aspect ratio of the rod and the tube**

**DP12 = Snap fit**

**DP13 = Tapered tip of the pin**

# **Decomposition of FR2 (Control contact resistance to be less than 20mΩ) and DP2 (Locally compliant electric contact)**

**FR21 = Prevent oxidation of the conductor**

**FR22 = Remove wear particles**

**FR23 = Control line tension/deflection of the non-conducting fiber**

**DP21 = Gold plated metal surface**

**DP22 = Space created in the crevices between fibers**

**DP23 = Spring**

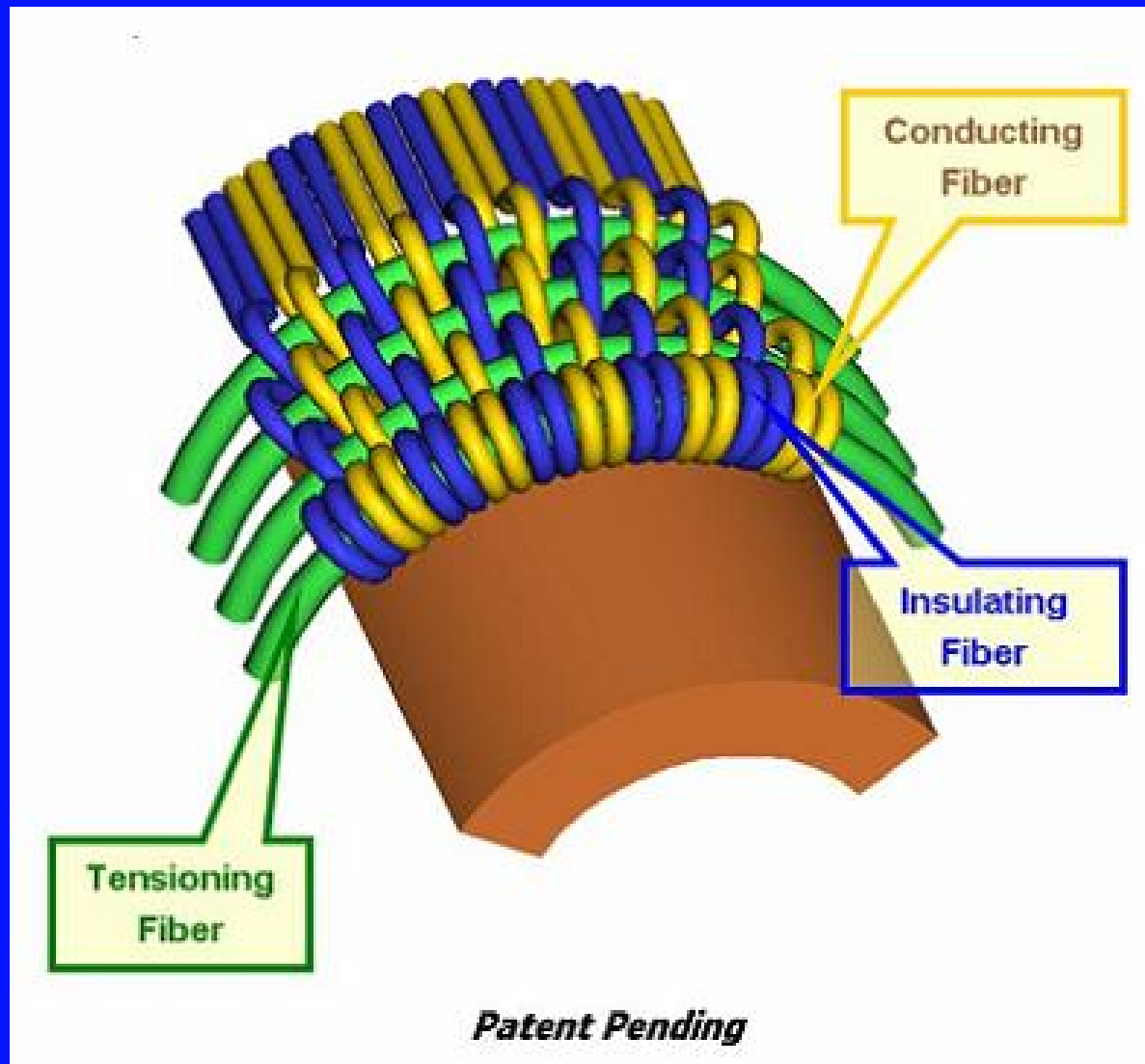


## Design Matrix

$$\begin{Bmatrix} FR1 \\ FR2 \\ FR3 \end{Bmatrix} = \begin{bmatrix} X & 0 & 0 \\ X & X & 0 \\ 0 & X & X \end{bmatrix} \begin{Bmatrix} DP1 \\ DP2 \\ DP3 \end{Bmatrix}$$

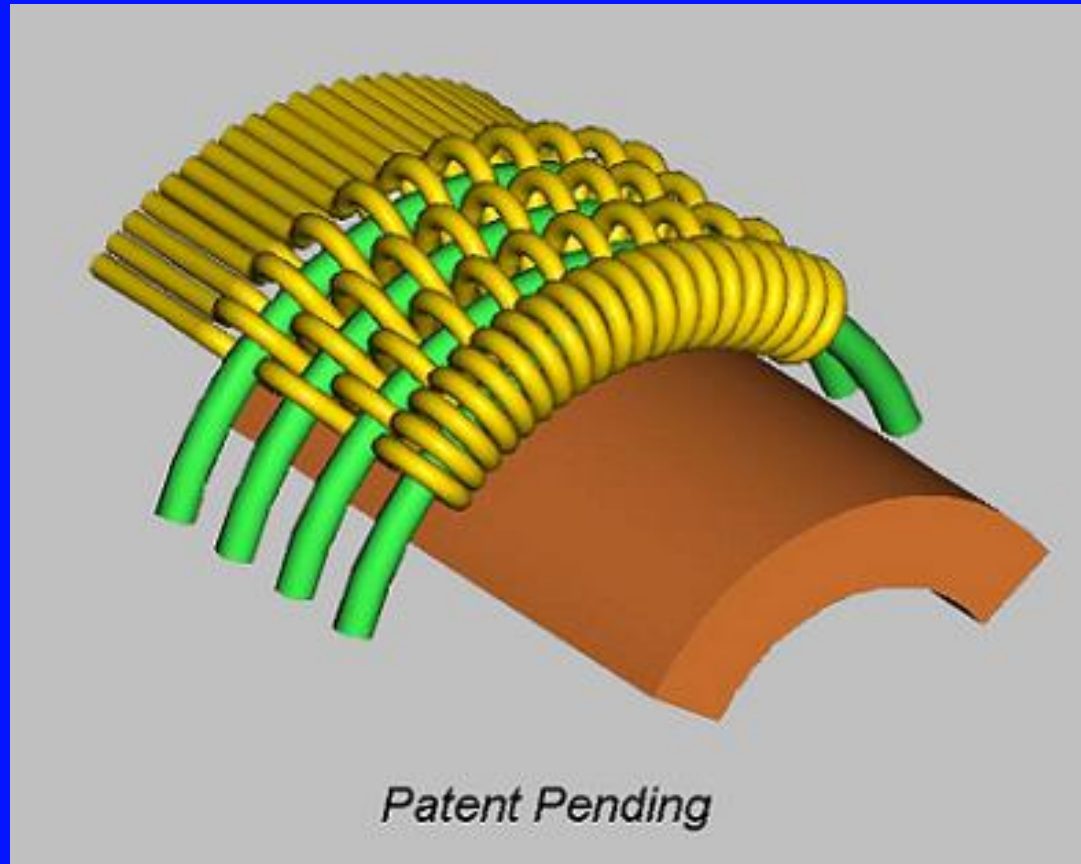
# Tribotek Electrical Connectors

(Courtesy of Tribotek, Inc. Used with permission.)



# Tribotek Electrical Connectors

(Courtesy of Tribotek, Inc. Used with permission.)



# Performance of “Woven” Power Connectors

- **Power density  $\Rightarrow$  200% of conventional connectors**
- **Insertion force  $\Rightarrow$  less than 5% of conventional connectors**
- **Electric contact resistance = 5 m ohms**
- **Manufacturing cost**
- **Capital Investment**